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Steady-State Thermal Model of a Synchronous Reluctance Motor

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Abstract— This paper presents an analytical thermal model of a synchronous reluctance motor (SynRMs) to predict the temperature of its different parts. For the analytical calculation, a lumped parameter thermal network (LPTN) is developed and analyzed. To validate the developed model, measurements are carried out on an experimental setup including the analyzed machine. Finally, the test results are compared with the analytical results to assess the accuracy of the model.

Keywords—ac motors; lumped parameter thermal network; Synchronous reluctance motor; thermal analysis; thermal model

I. INTRODUCTION

Recently, there has been a growing interest in SynRMs due its low cost and simple structure as well as simple construction and manufacturing process [1]. This topology is presented as the right choice for traction and high-speed applications. A number of efforts has been done to achieve higher torque and power density, higher energy efficiency, and cost reduction. To achieve these goals, the thermal analysis of the electrical machine in parallel with the electromagnetic design is necessary.

The thermal analysis of an electrical machine is divided into two categories; analytical LPTN and finite element analysis (FEA) [2],[3]. The FEA has an ability to model complex shapes as well as accurate conduction heat transfer and heat loss distribution inside solid materials [2], [4] However, it is a time-consuming method and the convection boundaries are evaluated by means of the same empirical and analytical correlations which are applied for the LPTN[4], [5].

Some advantages of LPTN compared to the FEA are the fast calculation time with an acceptable accuracy. The thermal designers of electrical machines apply this method to evaluate the temperature of the key components of the electrical machine. There is a number of reports for LPTN analysis of the electrical machines [5]-[10]. Most of these reports deal with LPTN of the induction [6]-[9] and permanent magnet synchronous machines [5],[10]. In some models e.g., the proposed thermal model in [6], the heat transfer paths have

been developed in both radial and axial directions. As a result, the network complexity in terms of model details such as the number of nodes and thermal resistances and the calculation time has been increased. In order to tackle the mentioned problems, Boglietti in [9], presented a simplified thermal network for an induction motor. Accordingly, the number of the thermal nodes has been reduced to six and the axial heat transfer paths have been confined to the shaft and the end-windings.

There has been a small number of reports describing the thermal model of the SynRMs. e.g., in [11], Rasid mainly focused on the heat transfer inside the stator slots and proposed a thermal model for the active part of the machine. In [1], Boglietti used the FEA for modeling these active parts of the machine.

This paper presents a simplified LPTN for a SynRM based on the proposed thermal model in [9]. This model has been specially developed for steady-state thermal analysis of a totally enclosed fan cooled (TEFC) transverse- laminated SynRM.

Fig. 1. The cross-section of the 11-kW transverse-laminated SynRM analyzed in this paper.

II. THERMAL MODEL DETAILS

A. Thermal Model Equivalent Circuit

The thermal model developed in this paper is intended for a four poles 10 kW, 400 V, 50 Hz, transverse-laminated SynRM with F insulation class, which its cross-section is shown in Fig. 1. Table I and II show the geometrical and material data of the experimental SynRM.

TABLE I. GEOMETRICAL DATA OF TRANSVERSE- LAMINATED SYNRM

Name	Unit	Value
Stator core length	mm	156
Stator inner diameter	mm	136
Stator outer diameter	mm	219
Number of slots	-	36
Air-gap height	mm	0.4
Rotor inner diameter	mm	45
Rotor outer diameter	mm	135.2
Coil height	mm	21
Slot filling factor	-	0.6
Slot area	mm ²	130.1

TABLE II. MATERIAL DATA OF TRANSVERSE- LAMINATED SYNRM

Machine part	Material	Thermal conductivity (W/mK)
Frame	Aluminum	230
Laminations	Electric steel	28
Winding	Copper	387
Impregnation	Resin	0.2
Air gap	Air	0.0257
Shaft	Steel	41

To model the steady-state thermal behavior of the machine with the LPTN method, the different heat transfer paths and thermal losses of the machine are represented by means of impedances and power sources respectively. During this process, we implement some hypotheses to reduce the complexity of the thermal model. E.g., the heat losses distribution is assumed to be uniform, the heat flux can only transfer in the axial direction through the shaft, end-winding and end regions of the machine, the heat flux distribution in both end regions of the machine are analogous.

Figure 2 presents the developed LPTN of the machine. The thermal model consists of five nodes and 13 thermal resistances, which capture all the key thermal parameters and temperature raises as well as the main thermal heat transfer paths. Table III gives the definition of the thermal model components of Fig. 2.

Since the SynRM has the same stator structure as an induction machine, the only difference between the developed thermal model in Fig. 2 with the thermal model of the induction machine proposed in [9] is the rotor model. Precisely, the only thermal resistances which will be different from the proposed model in [9] are R_{10} and R_{11} . The detailed evaluation methods of the other thermal resistances in Table III can be found in [9] and are not described here.

Fig. 2. The developed LPTN model of SynRM.

TABLE III

DEFINITION OF THERMAL MODEL COMPONENTS

Component	Description
R_0	Natural Convection and radiation thermal resistance from frame surfer to ambient
R_1	Interface gap conduction thermal resistance between the frame and active part of the machine
R_2	Conduction thermal resistance of the upper half side of the stator yoke
R_3	Conduction thermal resistance of the lower half side of the stator yoke
R_4	Conduction thermal resistance of the stator teeth
R_5	Conduction thermal resistance between the stator winding and stator teeth
R_6	Conduction thermal resistance between the stator winding and frame
R_7	Convection thermal resistances between the stator end-winding and inner air of end region
R_8	Convection thermal resistance between the inner air and end cap
R_9	Convection thermal resistance of the air gap
R_{10}	Conduction thermal resistance of upper half side of the rotor
R_{11}	Conduction thermal resistance the lower half part of the rotor
R_{12}	Axial thermal conduction of the shaft
P_{ir}	Stator iron losses
P_J	Stator joule losses
P_e	Rotor iron losses

Fig. 3. Cross section of 1/8 of the rotor.

B. Calculation of Rotor Thermal Resistances

Figure 3 shows the geometry of a half a pole of the SynRM rotor. The rotor construction consists of the iron laminations, which makes the magnetic flux paths and air gaps in the laminations, which are the magnetic flux barriers. By using the hypothesis that the heat transfer inside the flux barriers is carried out through natural convection and also due to the high thermal conductivity of the rotor iron laminations; we eliminate the heat transfer by the natural convection phenomenon from magnetic flux barriers. As a result, the total heat fluxes inside the rotor are transferred from the iron laminations.

Fig. 4. The thermal equivalent circuit of the rotor.

Figure 4 shows the equivalent thermal circuit for the calculation of the total thermal resistance from the rotor surface to the shaft, which is illustrated by R_r . Furthermore, the value of R_r is equal to the sum of the values of R_{10} and R_{11} . Accordingly, R_r consists of the series-parallel combinations of thermal resistances which the values of thermal resistances R_{11r} to R_{15r} are calculated by (2) and the rest are evaluated by (1). Finally, by using the series-parallel resistances law in the electrical circuit the value of R_r is calculated.

$$R = \frac{l}{\lambda \cdot A}, \quad (1)$$

$$R = \frac{\ln\left(\frac{r_o}{r_i}\right)}{\lambda \cdot L \cdot \phi}, \quad (2)$$

where l is the length of heat flux path (m), A is the cross-section area (m²), λ is the thermal conductivity of the material (W/m.K), r_o and r_i are the outer radius and the inner radius of cylinder (m), L is the active length of the machine (m) and ϕ is the angular span (rad).

C. Analytical Nodal Temperature Calculation

For the steady-state thermal analysis, the final nodal temperatures of Fig. 2 can be calculated by the matrix inversion theory [12]. Accordingly, the nodal temperatures of the proposed thermal model are calculated as follow:

$$[T] = [G]^{-1} [P], \quad (3)$$

where $[T]$ is the temperature column vector, $[P]$ is the power column vector which contains the losses at each node and $[G]$ is the thermal conductance square matrix which is defined as:

$$G = \begin{bmatrix} \sum_{i=1}^n \frac{1}{R_{1,i}} & \frac{-1}{R_{1,2}} & \dots & \frac{-1}{R_{1,n}} \\ \frac{-1}{R_{2,1}} & \sum_{i=1}^n \frac{1}{R_{2,i}} & \dots & \frac{-1}{R_{2,n}} \\ \vdots & \vdots & \ddots & \vdots \\ \frac{-1}{R_{n,1}} & \frac{-1}{R_{n,2}} & \dots & \sum_{i=1}^n \frac{1}{R_{n,i}} \end{bmatrix} \quad (4)$$

The $G_{i,i}$ components in the main diagonal of the thermal conductance matrix are defined as the sum of the conductances connected to the i -th node and $G_{i,j}$ is defined as the negative thermal conductances between nodes i and j .

III. EXPERIMENTAL METHODOLOGY

The objective of the experimental work in the natural cooling mode is to determine the value of the natural convection and radiation thermal resistance (R_0) as well as the accuracy of the analytical calculation based on the LPTN model.

According to [9], the value of the resistance R_0 is evaluated by using DC stator test. The DC stator test is a common experimental method for determining the value of the resistance R_0 . In this test, the loss of SynRM is confined to the joule loss of the stator coil windings where the electric power can be easily measured. In the calculations, we accounted for the variations in the winding electrical resistance, as the winding resistivity is temperature dependent [12],[13]. During the experiments, the DC power applied to the motor is measured as well as the surface temperature of the motor at different locations. The SynRM is supplied through a digital DC power supply, since the forced cooling air speed is zero and the motor is cooled by natural convection, the motor is supplied with 40% -70% of its nominal current for overheat protection. Four K-type thermocouples are installed by means of adhesive material in various locations on the frame surface of the motor. The ambient temperature is also measured by means of a K-type thermocouple. For the purpose of increasing the accuracy of the temperature measurement and minimizing the contact resistance between the thermocouple and the frame surface of the motor, we used thermal paste. The average temperature of these four thermocouples is assumed to be the mean temperature of the motor frame surface. During the experiments, all the temperature data are collected by means of a Graphtec GL200. The experiment has been carried out until the system reached its steady-state condition. The resistance R_0 is then calculated as [14]:

$$R_0 = \frac{T_s - T_a}{P}, \quad (5)$$

where T_s is the frame surface temperature ($^{\circ}\text{C}$), T_a is the ambient temperature ($^{\circ}\text{C}$) and P is the input DC power (W).

Fig. 5. SynRM test setup.

Figure 5 shows the experimental setup and different location of the surface frame thermocouples. In addition to the

surface temperature, the temperature inside the machine was also measured. For this purpose, six different thermocouples have been installed inside the end-windings and slots of the stator. Finally, all measured temperature values are used to validate the developed thermal model.

IV. RESULTS AND DISCUSSION

The DC measurements were carried out at current equals to 40% of the nominal current of the machine, i.e. 8.71 (A). It then took about seven hours to reach the steady-state thermal behavior.

In the steady state mode, the input power was 191.16 (W), the mean temperature of the frame surface was 61.2 ($^{\circ}\text{C}$) and the ambient temperature was 21.8 ($^{\circ}\text{C}$). Finally, the evaluated value of R_0 by using (5) was 0.2056 ($^{\circ}\text{C}/\text{W}$).

TABLE IV

Analytical and Experimental Temperature of Motor Components		
Machine components	Analytical thermal model	Experimental mean temperature
Frame surface ($^{\circ}\text{C}$)	59.4	61.2
Winding ($^{\circ}\text{C}$)	70.8	74
End winding ($^{\circ}\text{C}$)	76.7	78

To validate the developed thermal model, the analytically calculated temperatures are compared with the experimental data. We present the analytical and experimental results in Table IV. Accordingly, the analytical results are in good agreement with those obtained by the experimental method. The mean difference in the temperature between the analytical and experimental results is around 3%. This difference arises from the hypothesis of the uniform distribution of losses in the machine as well as from the fact that the analytical LPTN model predicts the mean temperature of the machine component while in the experimental part, the thermocouples show the temperatures at given positions. This fact is illustrated in Fig. 6, which shows the temperature distribution over the frame surface of the SynRM in the steady-state mode. The picture is captured by a thermal camera. The figure gives a good overview of the temperature difference on the frame surface of the SynRM. however, The result shows that the presented thermal model can be applied for temperature perdition of the SynRM.

Fig. 6. The surface temperature of SynRM.

V. CONCLUSION

A steady-state LPTN of a transverse-laminated SynRM has been presented. It makes it possible to predict the temperature of the important parts of the machine. We have implemented some simplification hypothesis to reduce the complexity of the model to decrease the computation time without compromising the accuracy of the temperature prediction.

The experimental results of the SynRM under DC stator test have been compared to the LPTN predicted temperatures. The agreement between the analytical temperature and the measured ones seems to be very good. For the future job, the thermal model of the machine will be adjusted to predict the temperature of the machine under forced cooling and normal operation of the machine. For this reason, a radiation thermal resistance and a forced convection thermal resistance parallel to the natural convection one will be in the thermal model.

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